設施栽培機械化與自動化管理研習班

數位化溫室工程

- 1. 太陽軌跡與能量
- 2. 濕空氣熱力特性
- 3. 溫室通風與成本
- 4. 蒸發降溫系統



- 5. 温室供暖
- 6. 二氧化碳施肥
- 7. 植物蒸散模式

溫室供暖八法

其中

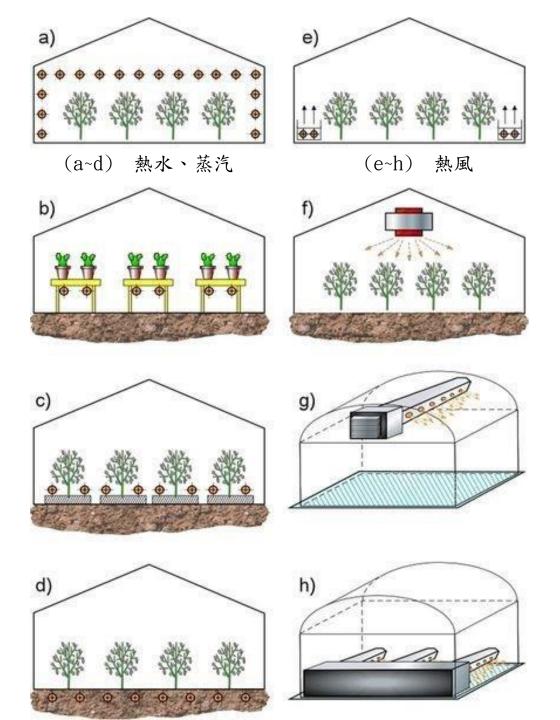
(d) Porous concert floor heating system 搭配

太陽能熱水系統

與

雙層充氣塑膠布溫室

在70年代,能源危機期間 幫美國溫室業者節省了大量的加溫成本





MATLABTM BASED SOFTWARE FOCUSING ON FLOOR HEATING

Wei FANG, Ph.D., Professor Dept. of Biomechatronics Engineering National Taiwan University

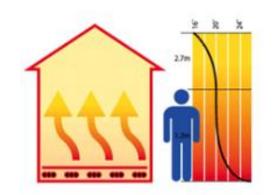
FROM GREENHOUSE TO HOUSE





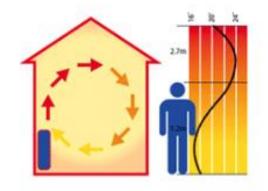
Radiant heat

Underfloor heating is the closest to an ideal heating temperature.



Convected heat

Radiators heat a room inefficiently, circulating heat in a spiral effect that creates cold spots.



FROM GREENHOUSE TO HOT YOGA



TO START

Unzip 'heating.zip' and copy all files to one directory, say: 'heating' Assign 'working directory' of MATLAB to the 'heating' directory.

Enter 'heating' in the command window of MATLAB, press 'Enter'

One Window with a menu on top and the opening page of the software introducing the model developed.

 \times

Greenhouse Heating v.4.0

Dept. of Bio-resource Engineering

George H. Cook College

Rutgers - The State University of New Jersey, U.S.A

Software developed by Wei Fang, Ph.D., Professor Dept. of Biomechatronics Engineering National Taiwan University

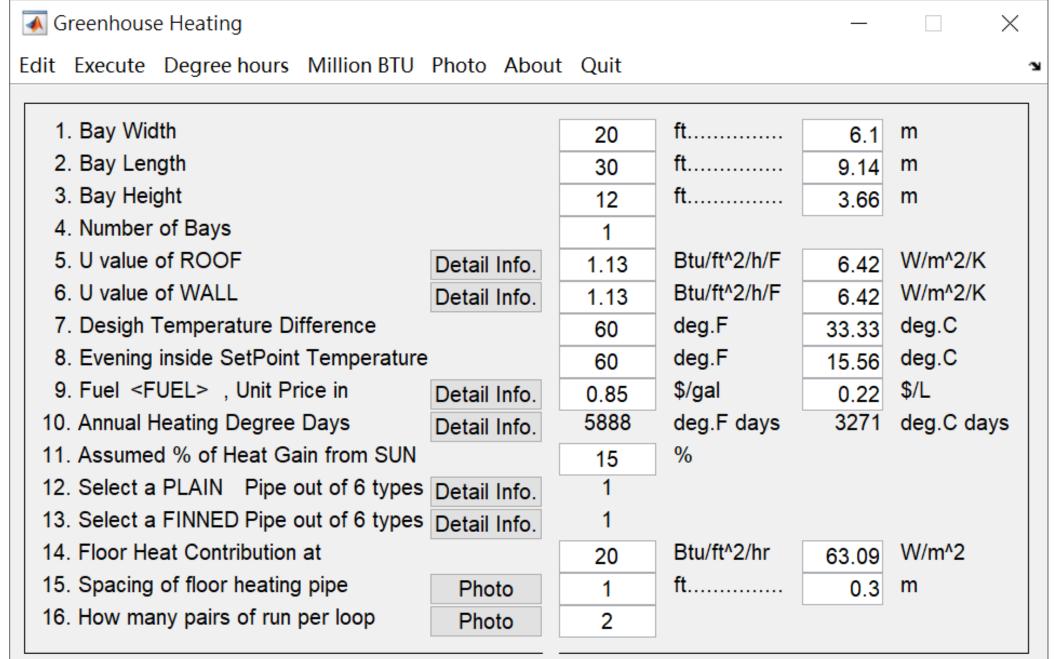
> v.3.0 Updated 02/04/2004 v.4.0 Last updated: 12/30/2020



After confirming all the input values, 'Execute' is now activated.

Totally, 8

buttons can be pressed for further information.



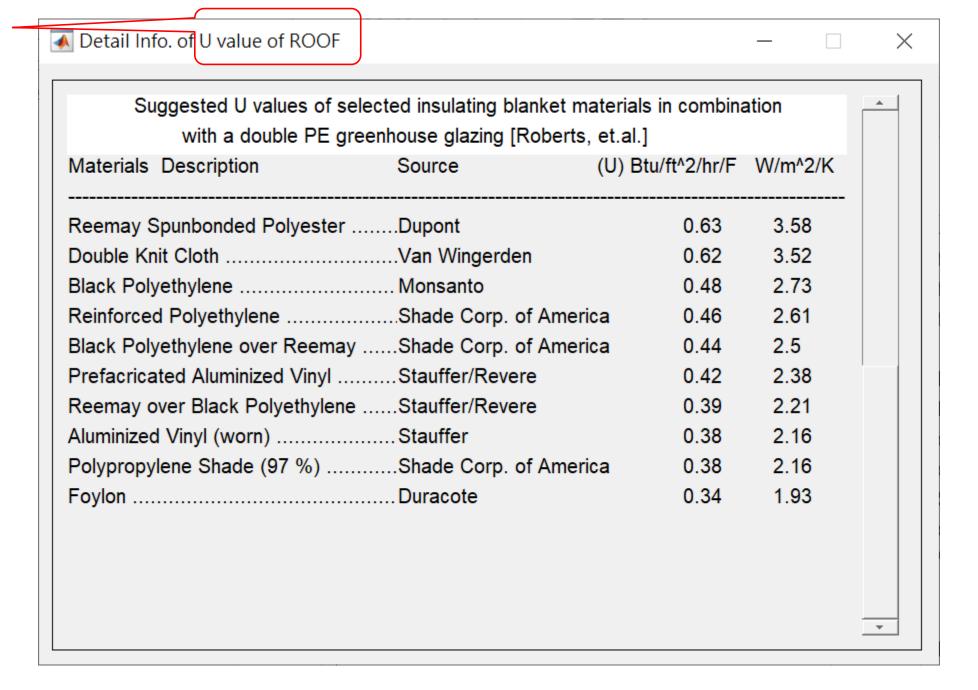
1st page of the first 'Detail info.' button.

Detail Info. of U value of ROOF

Covering Materials (Glazing)	Heat Loss Coef. (U) Btu/ft^2/hr/F	W/m^2/K	
Glass , single layer	1.13	6.42	
Glass , double-layer, 1/4-in (6 mm) space	0.65	3.69	
Glass , triple-layer, 1/4-in (6 mm) spaces	0.47	2.67	
Polyvinyl Chloride (PVC)	0.92	5.22	
Fiberglass-reinforced plastic (FRP)	1.2	6.81	
Acrylic, single layer 1/8-in (3 mm)	1	5.68	
Acrylic, double-layer pannel (16 mm thick)	0.58	3.29	
Acrylic, double-layer pannel (8 mm thick)	0.64	3.63	
Polycarbonate, double-layer pannel (16 mm	thick)0.58	3.29	
Polycarbonate, double-layer pannel (6.5 mm	n thick)0.69	3.92	
Polyethylene (PE) film, single layer (2,4,6 m	il)1.15	6.53	
Polyethylene (PE) film, double layer	0.8	4.54	
Polyester film, Mylar	1.05	5.96	
Polyvinyl fluoride (PVF), Tedlar, single layer	film1.05	5.96	
Polyvinyl fluoride (PVF), double-layer film	0.76	4.32	_

 \times

2nd page of the first 'Detail info.' button.



Page appear when the second 'Detail info.' button is pressed.

Detail Info. of U value of WALL

Suggested U value through various greenhouse gla		A / / A O / I /
Covering Materials (Glazing) Heat Loss Coef.	(U) Btu/ft^2/hr/F	vv/m^2/K
Glass , single layer	1.13	6.42
Glass , double-layer, 1/4-in (6 mm) space	0.65	3.69
Glass, triple-layer, 1/4-in (6 mm) spaces	0.47	2.67
Polyvinyl Chloride (PVC)	0.92	5.22
Fiberglass-reinforced plastic (FRP)	1.2	6.81
Acrylic, single layer 1/8-in (3 mm)	1	5.68
Acrylic, double-layer pannel (16 mm thick)	0.58	3.29
Acrylic, double-layer pannel (8 mm thick)	0.64	3.63
Polycarbonate, double-layer pannel (16 mm thick)	0.58	3.29
Polycarbonate, double-layer pannel (6.5 mm thick)	0.69	3.92
Polyethylene (PE) film, single layer (2,4,6 mil)	1.15	6.53
Polyethylene (PE) film, double layer	0.8	4.54
Polyester film, Mylar	1.05	5.96
Polyvinyl fluoride (PVF), Tedlar, single layer film	1.05	5.96
Polyvinyl fluoride (PVF), double-layer film	0.76	4.32

 \times

Page appear when the third 'Detail info.' button is pressed.

總共有4類型共1 7種燃料可以選擇

包括

6種燃煤

5種燃油

4種燃氣

2種木材

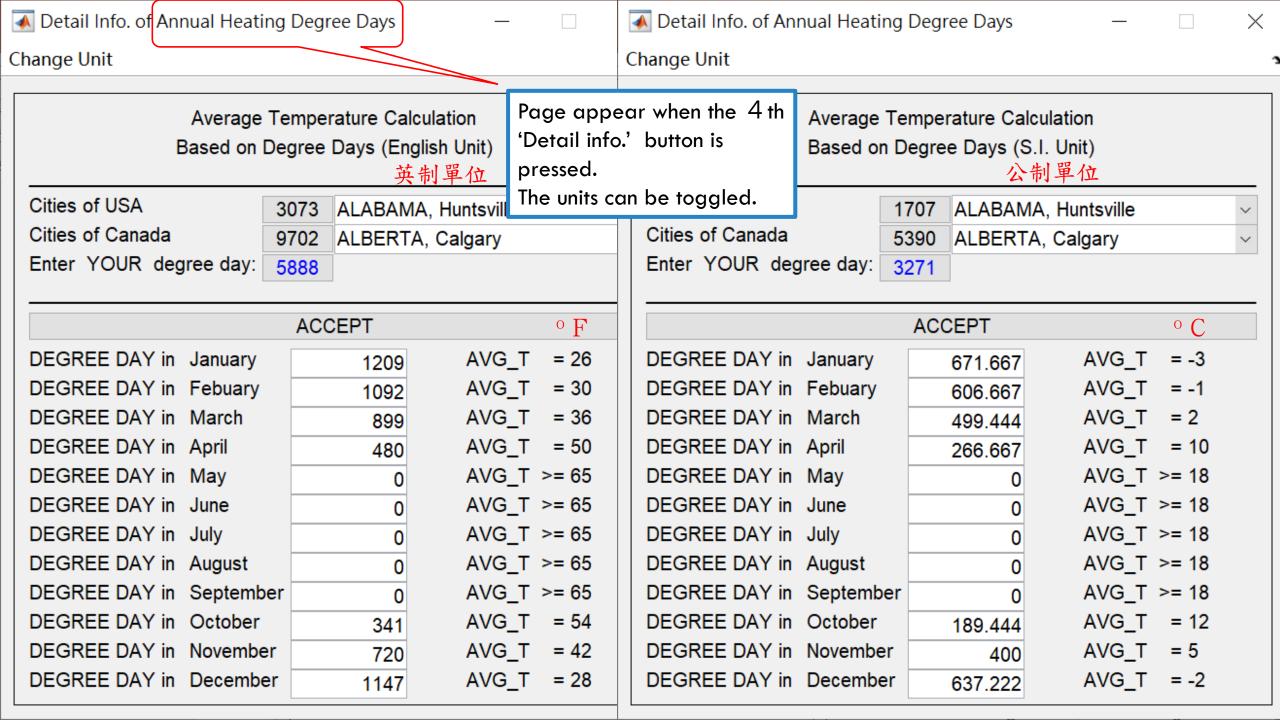
各種燃料的熱值可 有6種不同單位, 包含英制、公制與 業者常用單位

Tuel Type and Unit Price

[Typical Heat Contents for Various Types of Fuel Used for Greenhouse Heating]

X

			Heat	Value			Boiler Eff.
FUEL	Btu/lb	Btu/gal	Btu/ft3	kJ/g	kJ/ml	kJ/dm3	%
1. COAL, Anthracite (hard)	12910			30			65
2. COAL, Semi-anthracite	13770			32			60
 3. COAL, Low Volatile Bituminous 	14340			33.3			65
4. COAL, Medium V.B.	13840			32.2			60
5. COAL, High V.B.	11920			27.7			55
6. COAL, Sub-bituminous	9045			21.05			55
7. Fuel Oil No. 1		13495			37.65		70
8. Fuel Oil No. 2		13880			38.75		70
O 9. Fuel Oil No. 4		14695			41		68
10. Fuel Oil No. 5		15200			42.4		67
11. Fuel Oil No. 6		15335			42.8		65
12. Natural (Gas)		^	1000			37.3	75
○ 13. Manufactured (Gas)			550			20.5	70
14. Propane (LP Gas)		91690	2570			95.7	75
◯ 15. Butane (Gas)		10200	3225			120.1	75
○ 16. Wood, G. Green Chips	4500	^		10.5			60
17. Wood, H. Dried pellets	8500			19.8			60

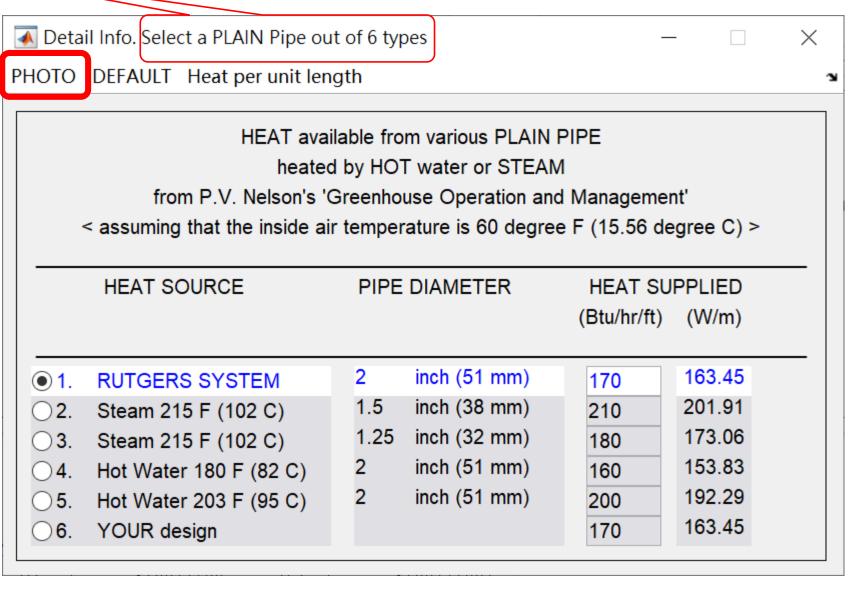


Page appear when the 5th 'Detail info.' button is pressed.

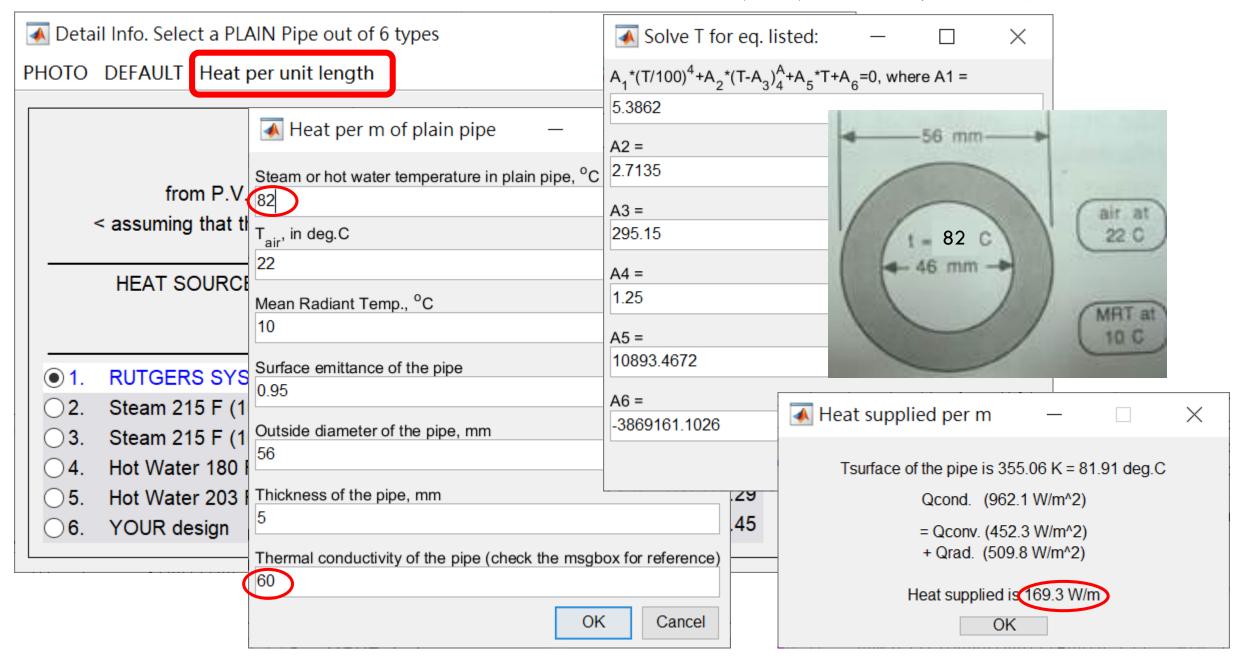


Photo depicts what is plain pipe

平滑的金屬管路經常安裝於溫室上方或植床下方



平滑的金屬管路每米長度的供熱能力計算



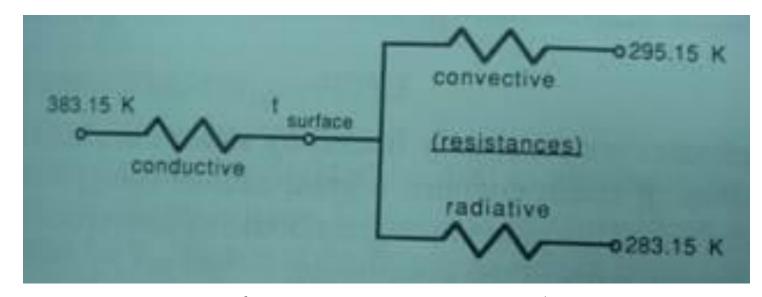
Material Thermal Conductivity, W/mK

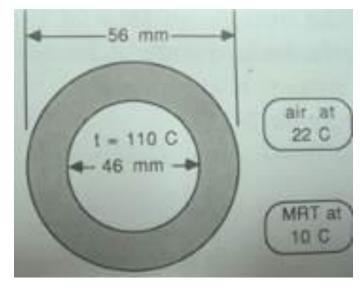
Metals			
Aluminum (alloy 110	00)	221	
Brass, red (85% Cu,	· ·	150	
Brass, yellow (65%	Cu, 35% Zn)	120	
Copper (electrolyti		393	,
Gold	*	297	
Iron, cast		47.7	(327 K)
Iron, wrought		60.4	
Lead		34.8	
Nickel		59.5	
Silver		424	
Steel, mild		45.3	
Tin		64.9	
Zinc. galvanizing		110	

計算範例

溫室的加熱方式中有一種是使用蒸汽通入金屬管,透過金屬管外表面與室內空氣的對流與輻射兩種熱傳方式進行加熱。假設金屬管外與內徑分別為 56 mm, 46 mm,金屬管本身的熱傳導係數(k) 為 52 W/mK. 加壓蒸汽的冷凝溫度為 110 ℃,假設此亦為金屬管的內表面溫度。溫室的室內空氣為22 ℃,溫室內加熱鐵管周遭的平均熱輻射溫度為 10 ℃。

假設金屬管表面塗成黑色,表面的熱放射率/輻射率(ε)為 0.95,請問金屬管與溫室之間的淨熱交換率是多少?其中有多少是對流,有多少是輻射?





對金屬管的表面而言,傳導過來的熱 = 對流 + 輻射流失的熱

依據圓柱座標熱阻公式

 $R_{conductive} = (\ln(r_o / r_i))/2\pi kL$ = $(\ln(28/23))/2\pi(52)$ (1) (per unit length) = 0.000602 mK/W.

每米長度有 0.1759 m² 的表面積

$$R_{\text{conductive}} = (0.000602 \text{ mK/W}) (0.1759 \text{ m}^2/\text{m})$$

= 0.000106 m²K/W,

透過熱傳導,傳到外表面的熱能計算如下:

$$q''_{conductive} = \Delta T/R$$

= (383.15 K - $T_{surface}$) / 0.000106 m²K/W.
= 9441(383.15 - $T_{surface}$)

加熱管輻射出的熱量假設全被溫室內空間吸收,透過輻射的熱損失計算如下:

$$q''_{radiative}$$
 = (0.95) (5.6697E - 8) ($T_{surface}^4$ - 283.154)
= 5.386($T_{surface} / 100$)⁴ - 346.3

由已知並無法確認管路外的對流是屬於層流或 是紊流。但可透過 Eq 3-34 的分析,計算如果 是紊流,所需要的溫差 (△)T

$$L^{3}\Delta T = 1$$
; $L = 0.056$ m. $\Delta T = 1/L^{3} = 5694$ K.

以溫室環境來看溫差不可能那麼大,所以一定 是層流,所以可選表 3-2 (p.65) 中公式 3-40 來 計算對劉熱傳係數 (h)

h = 1.32 ((
$$T_{surface}$$
 - 295.15 K) / 0.056 m)^{0.25},
= 2.713 ($T_{surface}$ - 295.15)^{0.25}.

由加熱管表面對溫室的對流熱傳遞計算如下:

$$q''_{convective} = h\Delta T = 2.713(T_{surface} - 295.15)^{1.25}$$
.

Conductive gain = Radiative loss + Convective loss

$$9441(383.15 - T_s) = 5.386(T_s/100)^4 - 346.3 + 2.713(T_s - 295.15)^{1.25}$$

 $5.386*(Ts/100)^4+2.713*(Ts-295.15)^{1.25}+9441*Ts-3617760=0$

求解得出Ts=382.9967K,代回上式,求出以下各項:

傳導: 1542 W/m^2 單位長度傳導的熱量為 $1542 * 0.1759 \text{ m}^2/\text{m} = 271.2 \text{ W/m}$

對流: 812 W/m² 輻射: 730 W/m²

MIXED MODE 的通式

$$A1*(Ts/100)^4 + A2*(Ts-A3)^{A4} + A5*Ts + A6 = 0$$

傳導 Conduction related parameters: A5, A6

對流 Convection related parameters: A2, A3, A4

輻射 Radiation related parameters: A1, A6

前兩例只有流體溫度與熱傳導係數不同,只影響熱傳導。 換言之,應該只有 A5, A6 有不同數值。

 $5.386*(Ts/100)^4+2.713*(Ts-295.15)^{1.25}+9441*Ts-3617760=0$

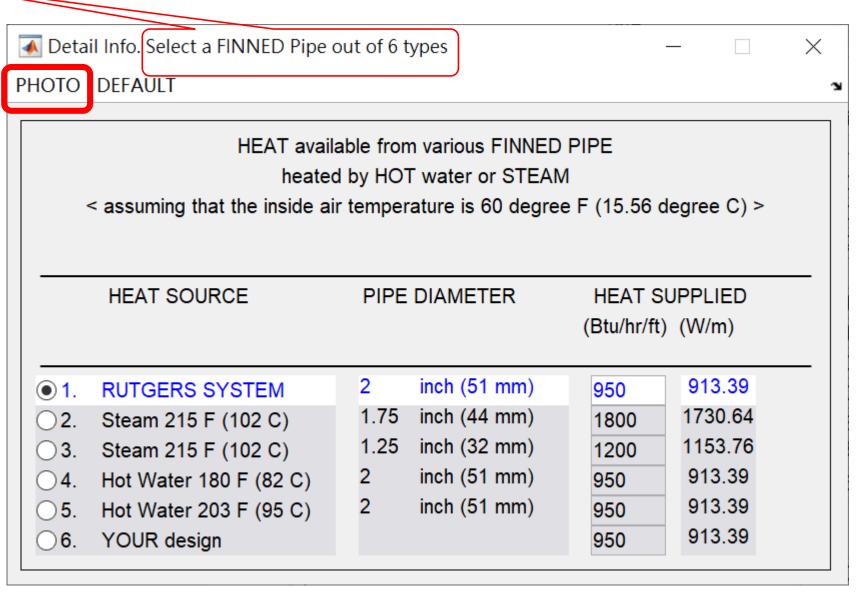
Solve T for eq. listed:	_		\times
A *(T/100) ⁴ +A */T A) ^A +A *T-	ιΛ =0 wh	oro A1 -	
$A_1^*(T/100)^4 + A_2^*(T-A_3)_4^A + A_5^*T$	6-0, wh	ere AT -	
5.3862			
A2 =			
2.7135			
A3 =			
295.15			
A4 =			
1.25			
A5 =			
10893.4672			
A6 =			
-3869161.1026			
			0 1
	(OK	Cancel

Page appear when the 6th 'Detail info.' button is pressed.



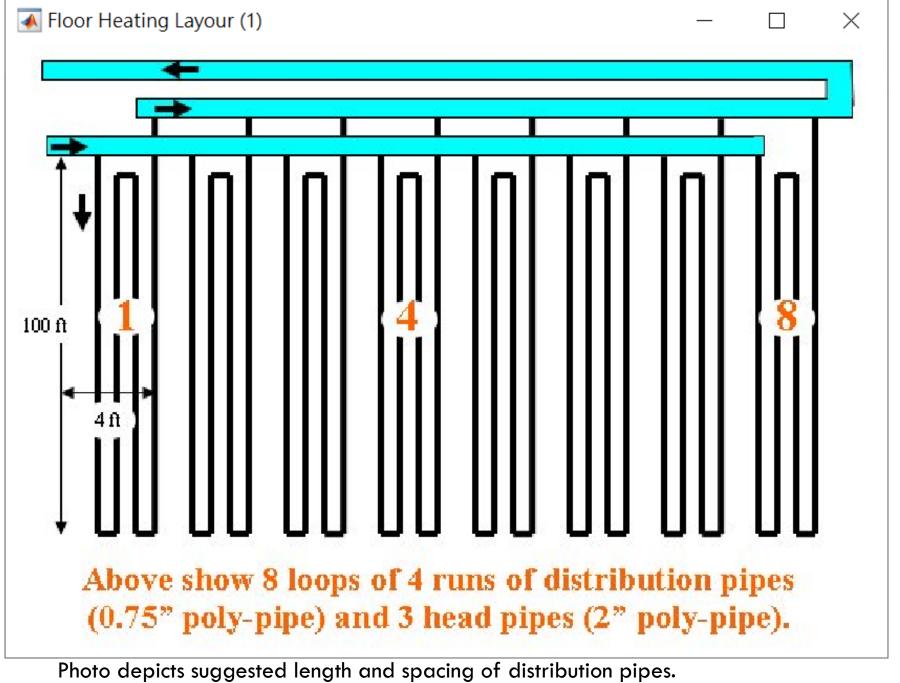
Photo depicts what is finned pipe

多鰭片的金屬管路 經常安裝於溫室靠外側的牆面





Photos depict what is 2 head pipes and 3 head pipes of a floor heating system



8 loops

2 pairs of

per loop

run = 4 runs

Photo depicts suggested length and spacing of distribution pipes.

Noted that 1 pair of run is 2 runs. Above shows 4 runs that is 2 pairs.

主管 Head pipe 2 吋塑膠管

分路管 Distribution pipe 0.75 吋塑膠管

8 loops 迴路

每議400室尺長個回 的限,100室尺長(兩 的限,的以有) 的限,因的所可對 (run)

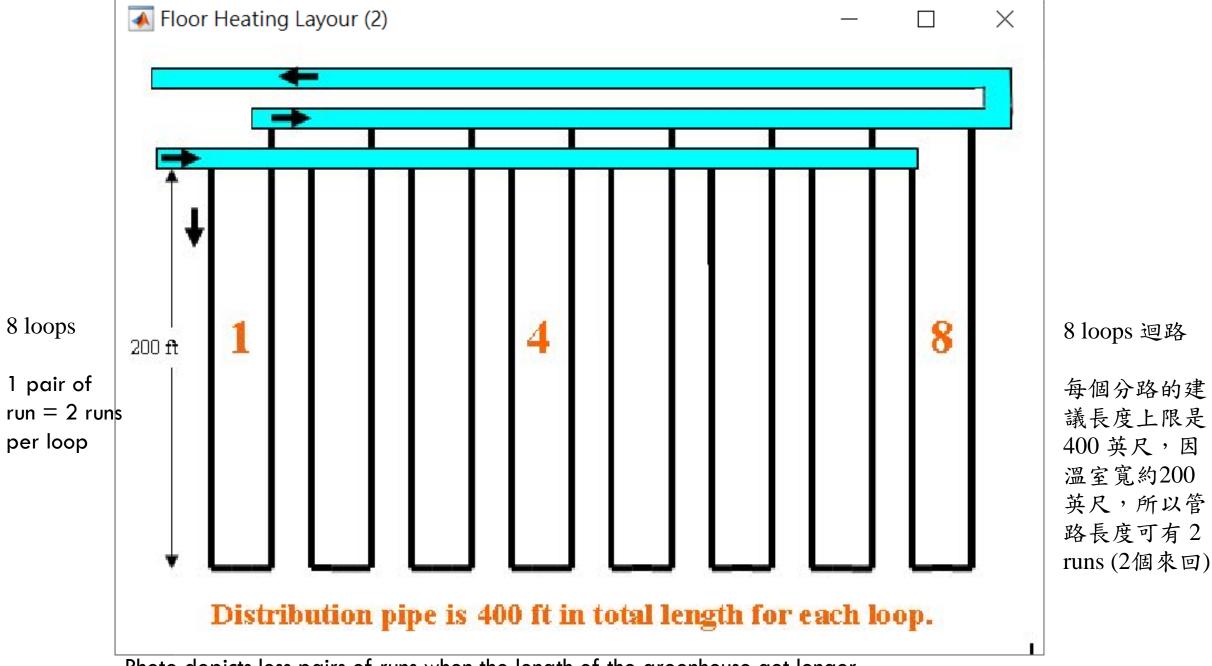


Photo depicts less pairs of runs when the length of the greenhouse get longer. In total, suggested length for the distribution pipe is 400 ft. This allows for only 1 pairs of runs.



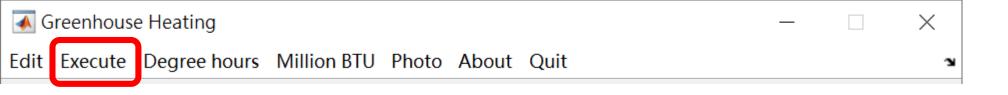
當溫室寬度達 400英尺時, 建議使用兩套 管路,每套使 用於200英尺 寬度。

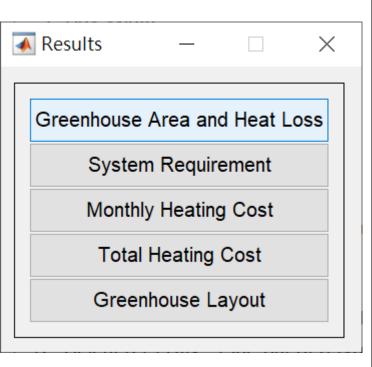
每個分路的建 議長度上限 400 英尺,因 溫室寬約200 英尺度的 所以管 Buns (2個來回)

2 zones

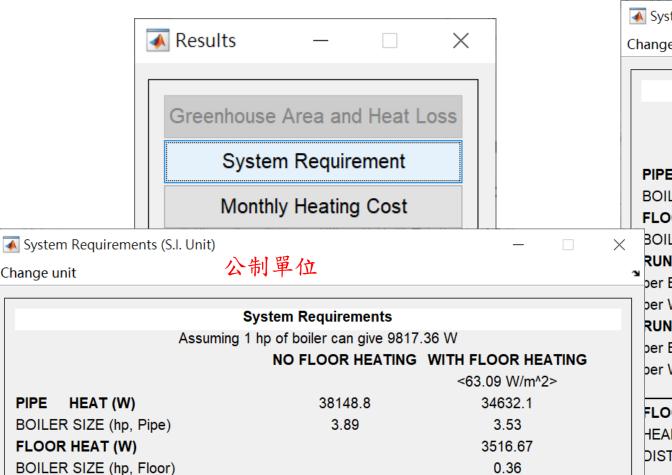
8 loops per zone

1 pair of run = 2 runs per loop per zone





,	se Area and Hea						<i>></i>
	Gre	enho	use Area and	Heat Loss			
ROOF	AREA	:	720	ft^2	66.89	m^2	
WALL	AREA	:	1200	ft^2	111.48	m^2	
SURFACE	AREA	:	1920	ft^2	178.37	m^2	
FLOOR	AREA	:	600	ft^2	55.74	m^2	
ROOF	HEAT LOSS	:	48816	Btu/hr	14305.8	W	-
WALL	HEAT LOSS	:	81360	Btu/hr	23843	W	
TOTAL	HEAT LOSS	:	130176	Btu/hr	38148.8	W	



9.57

4.79

1.71

0.86

20.12

192.02

7.22

4.79

1.29

0.86

m

m

Photo

Change unit

per BAY

per BAY

per WALL

per WALL

RUNS of PLAIN PIPE <163.45 W/m>

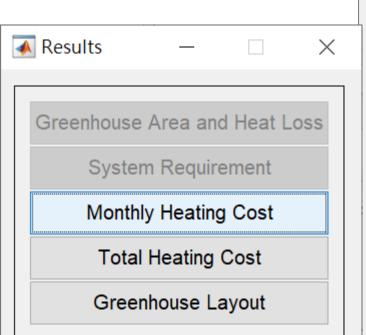
RUNS of FINNED PIPE <913.39 W/m>

HEADER PIPE (10% WASTE)

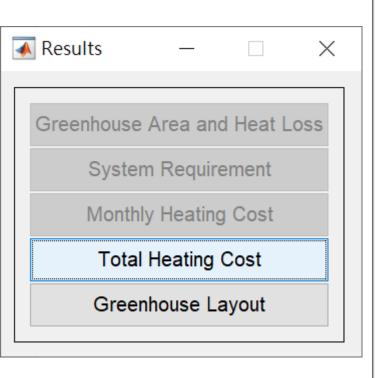
DISTRIBUTION PIPE (5% WASTE)

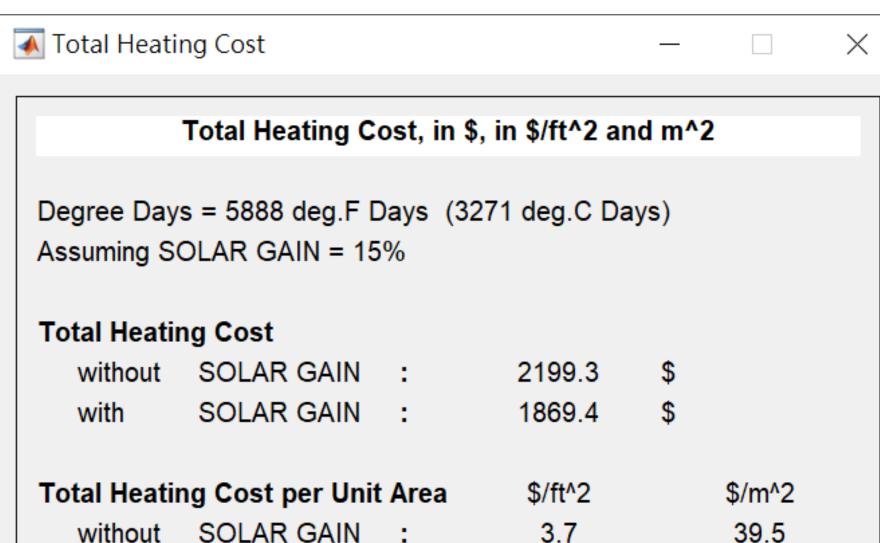
FLOOR PIPE LENGTH (3-head-pipe design)

承 System Requirements (English	Unit) 英制單位		X
Change unit	央刊千位		
	System Requirements		
Assur	ning 1 hp of boiler can give 335	00 Btu/hr	
	NO FLOOR HEATIN	G WITH FLOOR HEATING	
		<20 Btu/ft^2/hr>	
PIPE HEAT (Btu/hr)	130176	118176	
BOILER SIZE (hp, Pipe)	3.89	3.53	
FLOOR HEAT (Btu/hr)		12000	
BOILER SIZE (hp, Floor)		0.36	
RUNS of PLAIN PIPE <170 I	Stu/hr/ft>		
per BAY	9.57	7.22	
per WALL	4.79	4.79	
RUNS of FINNED PIPE <950	Btu/hr/ft>		_
per BAY	1.71	1.29	
per WALL	0.86	0.86	
FLOOR PIPE LENGTH (3-hea	d-pipe design)	Photo	
HEADER PIPE (10% WASTE)	66	ft	
DISTRIBUTION PIPE (5% WA	STE) 630	ft	



Monthly Heating Cost (in \$, \$/ft^2 and \$/m^2)									
Based on Degree Days, and 15% Solar Gain									
MONTH	single	double	single	double	d.PE w/	d.PE w/	yours		
	glass	glass	PE	PE	d.knit cloth	Foylon			
January	408.1	234.8	415.3	288.9	224.6	122.8	408.1		
Febuary	368.6	212	375.1	261	202.9	110.9	368.6		
March	288.1	165.7	293.2	204	158.6	86.7	288.1		
April	127.8	73.5	130	90.5	70.3	38.4	127.8		
May	0	0	0	0	0	0	0		
June	0	0	0	0	0	0	0		
July	0	0	0	0	0	0	0		
August	0	0	0	0	0	0	0		
September	0	0	0	0	0	0	0		
October	72	41.4	73.3	51	39.6	21.7	72		
November	220.7	127	224.6	156.3	121.5	66.4	220.7		
December	384.1	220.9	390.9	271.9	211.4	115.6	384.1		
Total, \$	1869.4	1075.3	1902.5	1323.5	1029	562.5	1869.4		
\$/ft^2	3.1	1.8	3.2	2.2	1.7	0.9	3.1		
\$/m^2	33.5	19.3	34.1	23.7	18.5	10.1	33.5		



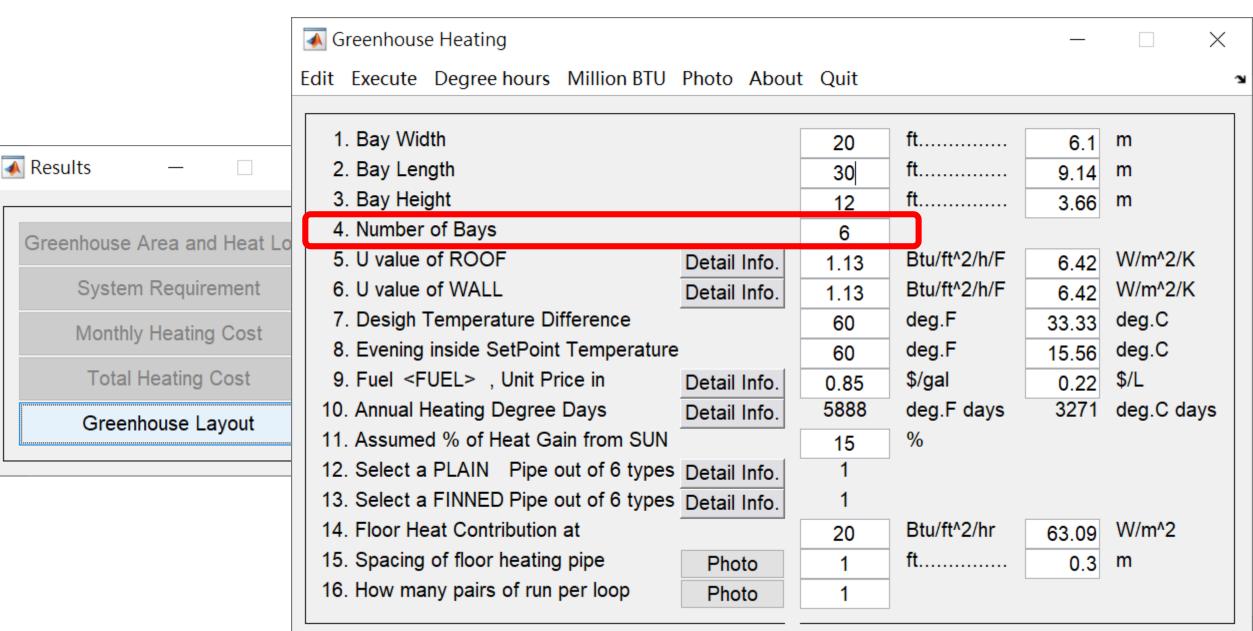


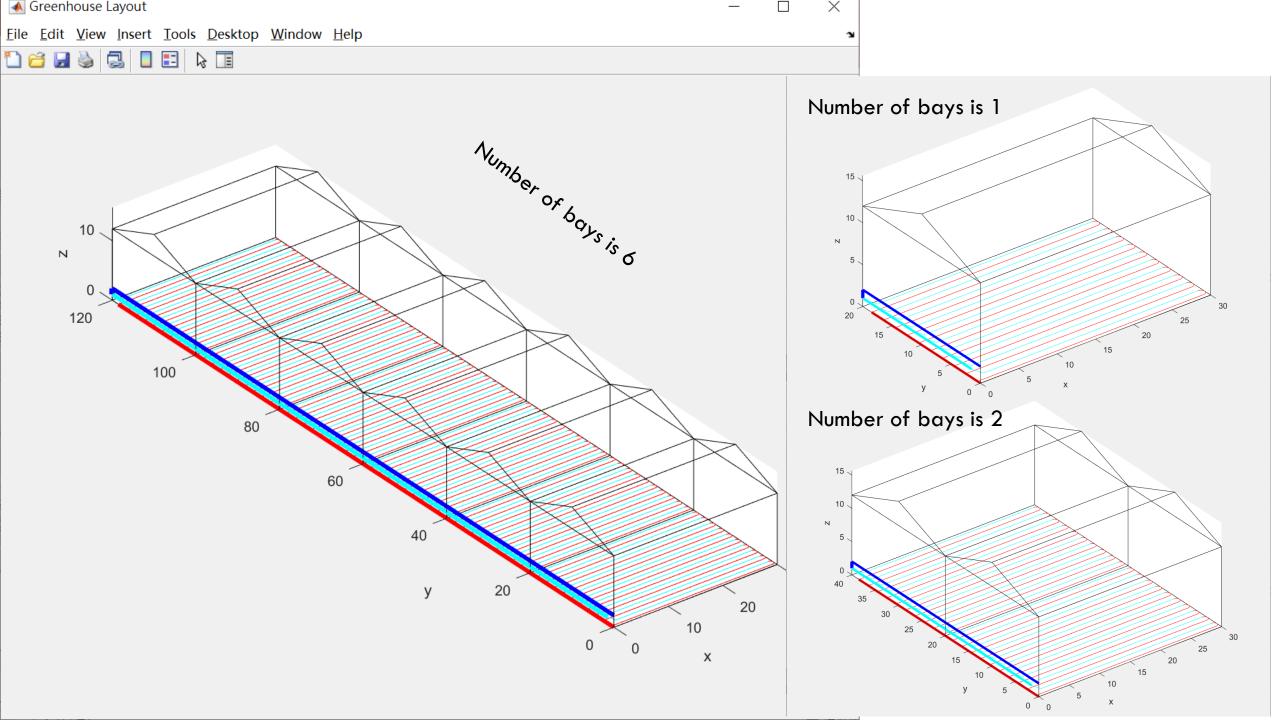
with

SOLAR GAIN :

3.1

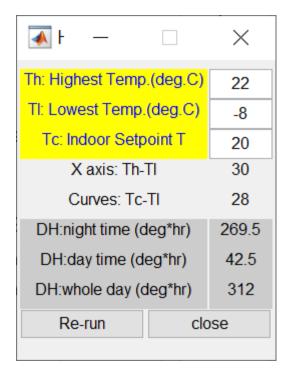
33.5



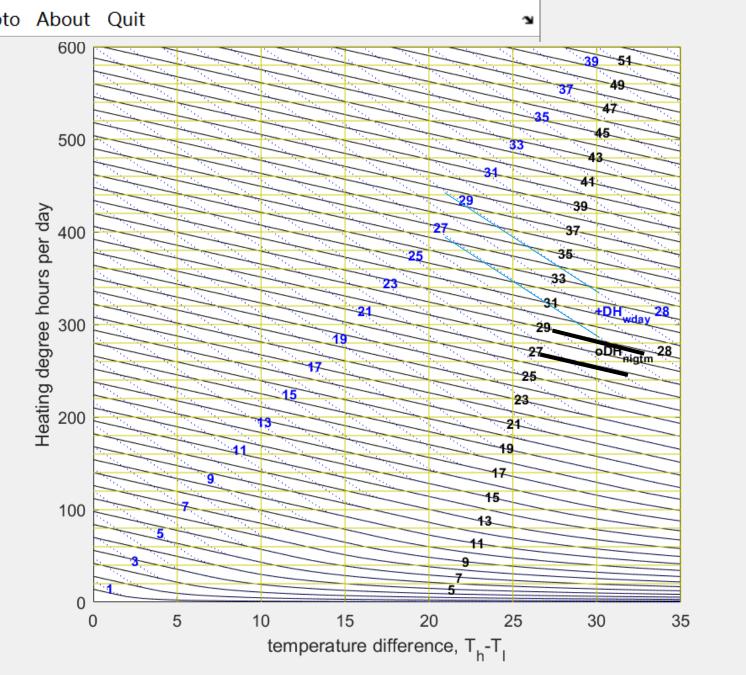




依據室外的最高、最低溫與室內的 設定溫度,推估加熱所需度時數



全日的加熱所需度時數(藍字)= 夜間度時數(黑字)+日間度時數





Edit Execute Degree hours

Million BTU Photo About

Quit

https://tc.unithelper.com/energy/

MillionBTU calculation for various types of fuel used in Boiler.

 \times

							/
Fuel Type	Heat '	Value	Boiler Eff.(%)	To reacl You	n MBTU nead	Marginal	Unit Price
1. COAL, Anthracite (hard)	12910	Btu/lb	6 5 ′		ton	105.8	\$/ton
2. COAL, Semi-anthracite	13770	Btu/lb	60	0.061	ton	104.17	\$/ton
3. COAL, Low Volatile Bituminous	14340	Btu/lb	65	0.054	ton	117.52	\$/ton
4. COAL, Medium V.B.	13840	Btu/lb	60	0.06	ton	104.7	\$/ton
5. COAL, High V.B.	11920	Btu/lb	55	0.076	ton	82.66	\$/ton
6. COAL, Sub-bituminous	9045	Btu/lb	55	0.101	ton	62.72	\$/ton
7. Fuel Oil No. 1	134950	Btu/gal	70	10.586	gal	0.6	\$/gal
8. Fuel Oil No. 2	138800	Btu/gal	70	10.292	gal	0.61	\$/gal
9. Fuel Oil No. 4	146950	Btu/gal	68	10.007	gal	0.63	\$/gal
10. Fuel Oil No. 5	152000	Btu/gal	67	9.819	gal	0.64	\$/gal
11. Fuel Oil No. 6	153350	Btu/gal	65	10.032	gal	0.63	\$/gal
12. Natural (Gas)	1000	Btu/ft^3	75	13.333	therm	0.47	\$/therm
13. Manufactured (Gas)	550	Btu/ft^3	70	25.974	therm	0.24	\$/therm
14. Propane (LP Gas)	2570	Btu/ft^3	75	518.807	therm	0.01	\$/therm
15. Butane (Gas)	3225	Btu/ft^3	75	4.134	therm	1.52	\$/therm
16. Wood, G. Green Chips	4500	Btu/lb	60	0.185	ton	34.04	\$/ton
17. Wood, H. Dried pellets	8500	Btu/lb	60	0.098	ton	64.3	\$/ton
<< <base fuel=""/> >>>	138800	Btu/gal	× 80	9.01	gal	0.7	\$/gal
基準燃料		熱值	效率				

以基準燃料的鍋爐效 率與燃料單價為基準, 當其他燃料的市價低 於表列的邊際單價時, 可以考慮以該種燃料 替换

有些設備可使用多種 燃料(1~6, 16, 17; 7~11; 12~15), 也有 的可切换使用不同燃 料譬如燃油或瓦斯 (7~15)

最常用的是編號8與14, 有些設備允許切換使 用,主要使用液化天 然氣(LP gas),但缺 料時可改用二號燃油

HEAT PUMP

Heat pump (HP) is a much more energy efficient system for heating, but It is also much more costly than the boiler.

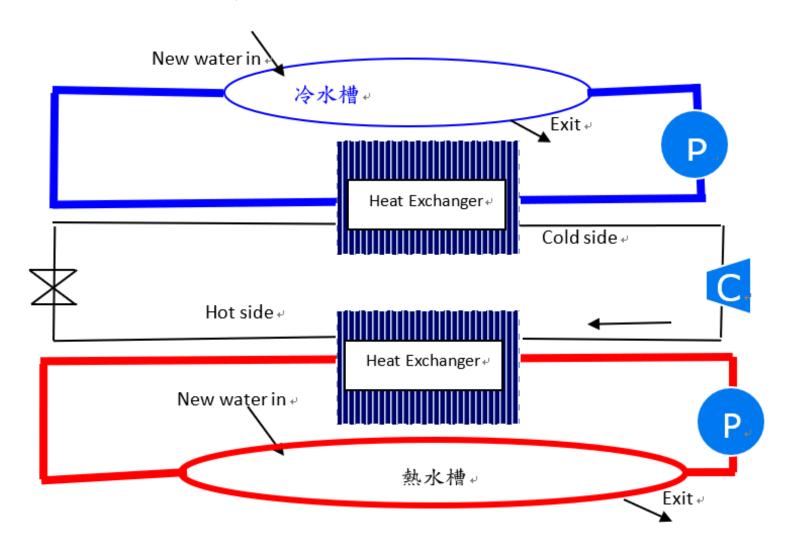
The heating cost of using HP can be calculated using following equation:

Heating cost@HP = Heating cost@boiler * eff_boiler / COP_HP

Assuming no.2 fuel was used. 1 MBTU (293 kWh) required 10 gal, the Boiler eff. Is 70% and the heating cost of the greenhouse assuming is $35 \ \$/m^2$

Assuming the COP of the HP is 4, that means consuming elec. 1 kWh can provide 4 kWh of heat. Compare with the efficiency of the burner, 0.7, the heating cost of using HP should be reduced to $35*4/0.7 = 6.14 \text{ } \text{/m}^2$

冷熱水雙效熱泵系統

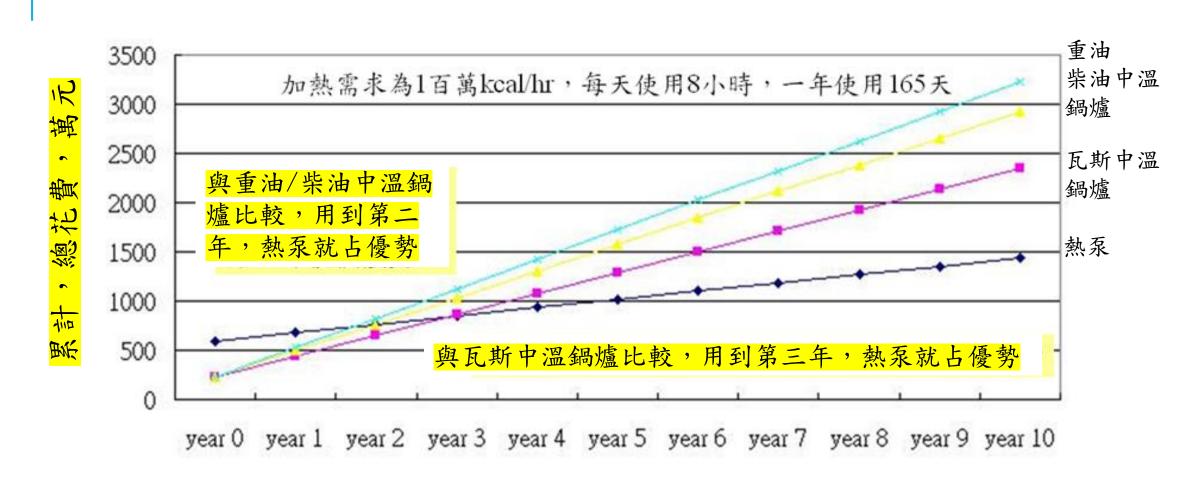


不同熱源加溫成本的比較

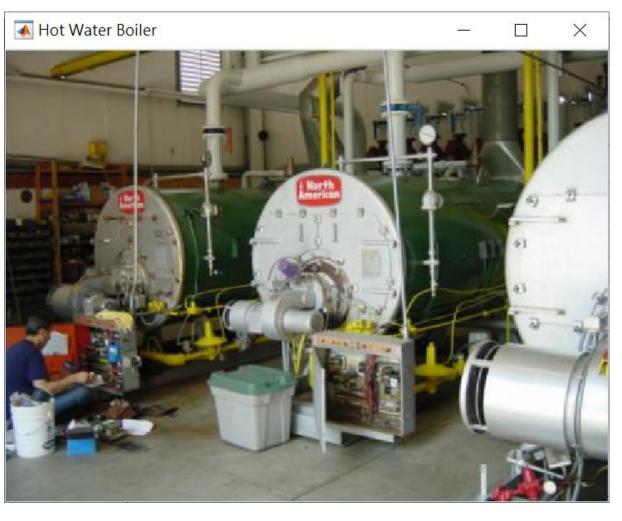
1564 kcal/元

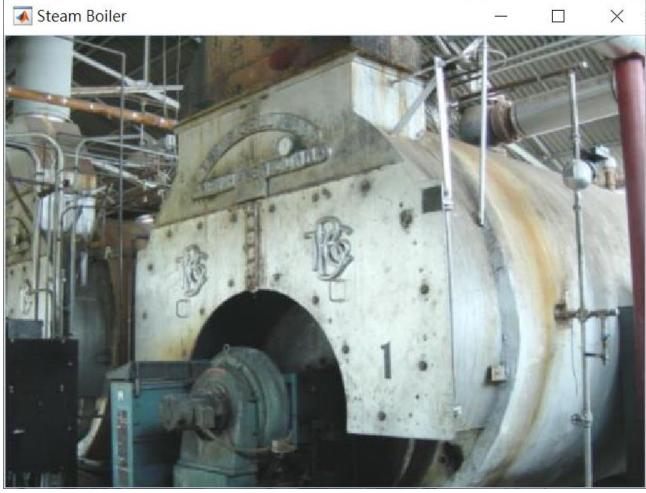


不同熱源系統設置與操作成本的比較

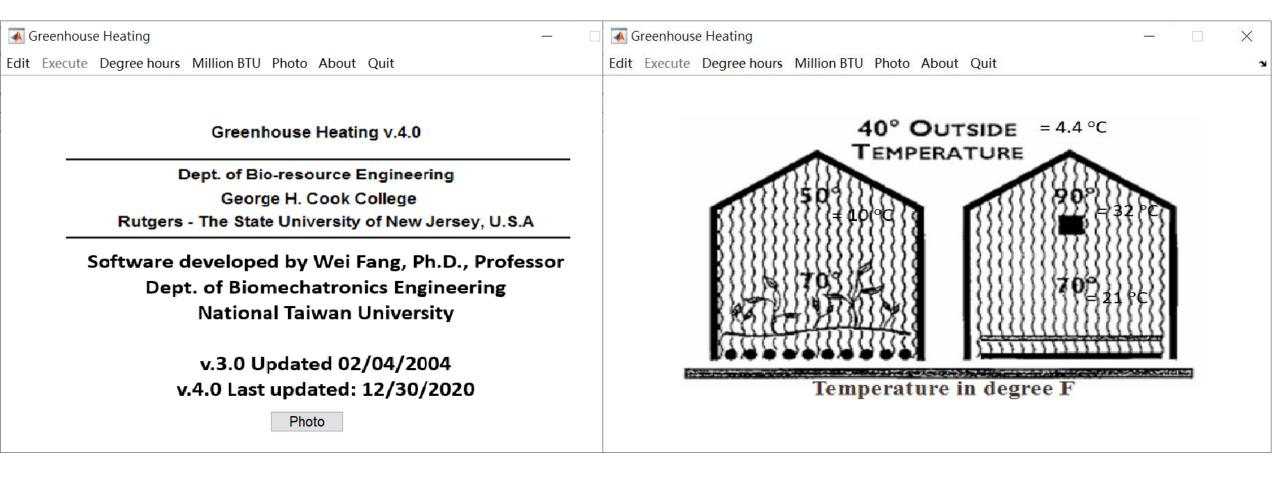


All the photos appeared previously can be found under this menu except these two.





Two more photos can be found under this menu. The first is the photo appeared in the opening page as shown below. The second shows the energy saving potential of the floor heating system.



CREDIT

The credit should go to researchers (Prof. William Roberts, David Mears, K.C. Ting, Gene Giacomelli and Mr. Tom Manning) at the Rutgers University when the author was a Ph.D. student in 1986.

The floor heating system was developed by Professor Roberts and colleagues back in 70s during the energy crisis. Together with the invention of air-inflated double-poly greenhouse, these two technologies help many growers a great deal to remove financial burden on heating.

The first DOS version program was developed by Prof. Roberts in 1980.

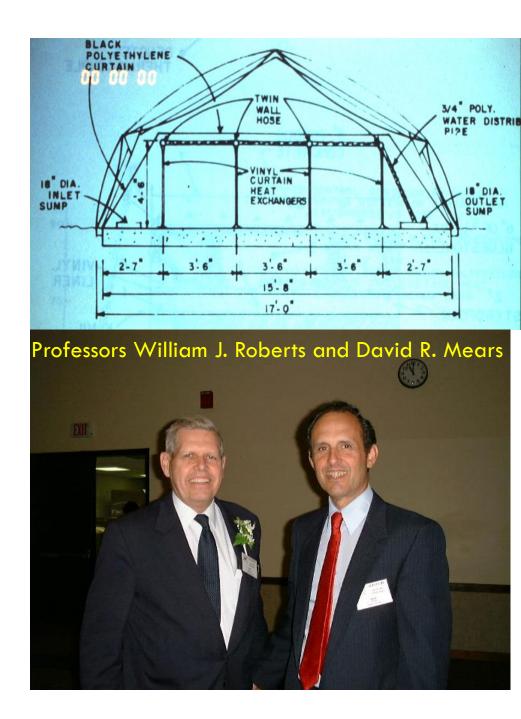
The previous Matlab version of the software was written in 2004. New version imported the concept of 'heating degree hour per day'.

Re-written of the software was to pay my respect to late Professor Roberts, who was RIP in 2020 during the COVID-19 pandemic.



雙層充氣塑膠布溫室的起源

- 1964年創於美國紐澤西州羅格斯大學 (Rutgers),40年後獲得ASAE 頒發超卓成就 獎牌並立碑紀念
- Roberts, W.J. and D.R. Mears. 1968.
 Double Covering a Film Greenhouse
 Using Air to Separate Film Layers. ASAE paper 68-402. 發表於 Logan, Utah.
- Published in Transactions of the ASAE V12,
 No.1 pp.32, 33, 38. 1969.



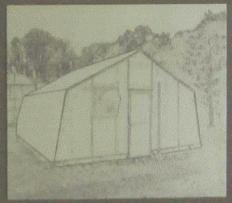


AN HISTORIC LANDMARK OF AGRICULTURAL ENGINEERING

A crucial step in the evolution of modern plant agriculture was the development of low-cost, energy-efficient greenhouse structures that provide optimum growing conditions year-round.

In 1964, Professor William J. Roberts developed the first airinflated double-layer polyethylene greenhouse covering system at Cook College, Rutgers University.

Air-inflated double-layer polyethylene greenhouse covering systems were quickly and widely adopted throughout the United States and across the world, primarily due to the relatively low installation costs, adequate light transmission, and significant insulating properties. Today, more than half of all the greenhouses worldwide are covered with the air-inflated double-layer polyethylene covering system.



DEDICATED BY THE
AMERICAN SOCIETY OF AGRICULTURAL ENGINEERS
2004



The 43rd Landmark of ASAE



充氣設計 需要導入 室外空氣 避免在雙 層塑膠布 內產生結 露

不僅單棟 温東 用 宝 温 標 可 景 温 点 樣 可 引 宝 可 以



ASSUMPTIONS OF THE 1980 MODEL

- A boiler provides 33500 BTU/h per HP
- 2. An extra 5% of Floor pipe length is needed for construction.
- 3. An extra 10% Header pipe with double return is needed.
- 4. Three Header pipes are needed, 1 supply and 2 return.
- 5. Floor pipes are spaced 1 foot (30 cm) center to center.
- 6. # 2 Oil has a heating efficiency of 70 %; therefore, the heat output is approximately 100,000 BTU/gallon.
- 7. The boiler size is determined based on the design temperature difference.
- 8. The heating system runs 24 hs/day at $(T_{setpoint} T_{outside})$
- 9. T_{outside} was determined based on the heating degree day of a specific location. The heating degree day calculation was based on 65 °F (18.3 °C).

CONTACT INFORMATION

The software was written by Professor Wei FANG for the purpose of teaching the Controlled Environment Agriculture related courses.

The software is free to distribute for education purpose only.

Any commands and/or suggestions are welcomed.

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